

quindi $\sum_{i=1}^n (m_i \vec{v}_i - m_i \vec{v}_G) = 0$ diventa $\sum_{i=1}^n (m_i \vec{v}'_i) = 0$

Riscriviamo le quantità:

$$1) \vec{Q} = \sum_{i=1}^n m_i \vec{v}_i = \sum_{i=1}^n m_i (\vec{v}_G + \vec{v}'_i) = (\sum m_i) \vec{v}_G + \underbrace{\sum_{i=1}^n m_i \vec{v}'_i}_{=0}$$

chiamiamo M la massa totale

$$\boxed{\vec{Q} = M \vec{v}_G} \quad \text{quantità di moto}$$

$$2) T = \frac{1}{2} \sum_{i=1}^n m_i (\underbrace{\vec{v}_G + \vec{v}'_i}_{\vec{v}_i}) (\vec{v}_G + \vec{v}'_i) = \quad \text{Th di König}$$

$$= \frac{1}{2} \sum_{i=1}^n m_i \vec{v}_G \vec{v}_G + \underbrace{\sum_{i=1}^n m_i \vec{v}'_i \vec{v}_G}_{=0} + \frac{1}{2} \sum m_i \vec{v}'_i \vec{v}'_i$$

$$\boxed{T = \frac{1}{2} M \vec{v}_G^2 + \frac{1}{2} \sum_{i=1}^n m_i \vec{v}'_i \vec{v}'_i} \quad \text{energia cinetica}$$

$$3) \boxed{\vec{K}_A = (\vec{G} - A) \times M \vec{v}_G + K_A'} \quad \text{momento angolare}$$

momento angolare di un punto che è concentrato in G e si muove di $\vec{v}_G$

$$K_A' = \sum_{i=1}^n m_i (\vec{p}_i - A)$$

OSS formula del trasporto per $\vec{K}_A$

$$\left[\begin{array}{l} \vec{K}_B = \vec{K}_A + \underbrace{\vec{Q}}_{M \vec{v}_G} \times (B - A) \quad (\text{atto moto rigido}) \\ \vec{K}_B = \vec{K}_A + (A - B) \times \vec{Q} \end{array} \right. \quad \vec{v}_B = \vec{v}_A + \omega \times (B - A)$$

Meccanica Razionale 2nd Edition Dinamica

Tullio Levi-Civita, Ugo Amaldi



Meccanica Razionale 2nd Edition Dinamica:

Lezioni di meccanica razionale ...: -2. pte. Dinamica dei sistemi con un numero finito di gradi di libertà Tullio Levi-Civita, Ugo Amaldi, 1926 Gears Vincenzo Vullo, 2020-01-24 The book explores the geometric and kinematic design of the various types of gears most commonly used in practical applications also considering the problems concerning their cutting processes The cylindrical spur and helical gears are first considered determining their main geometric quantities in the light of interference and undercut problems as well as the related kinematic parameters Particular attention is paid to the profile shift of these types of gears either generated by rack type cutter or by pinion rack cutter Among other things profile shifted toothing allows to obtain teeth shapes capable of greater strength and more balanced specific sliding as well as to reduce the number of teeth below the minimum one to avoid the operating interference or undercut These very important aspects of geometric kinematic design of cylindrical spur and helical gears are then generalized and extended to the other examined types of gears most commonly used in practical applications such as straight bevel gears crossed helical gears worm gears spiral bevel and hypoid gears Finally ordinary gear trains planetary gear trains and face gear drives are discussed Includes fully developed exercises to draw the reader's attention to the problems that are of interest to the designer as well as to clarify the calculation procedure Topics are addressed from a theoretical standpoint but in such a way as not to lose sight of the physical phenomena that characterize the various types of gears which are examined The analytical and numerical solutions are formulated so as to be of interest not only to academics but also to designers who deal with actual engineering problems concerning the gears **Relativistic Theories of Materials** A. Bressan, 2012-12-06 The theory of relativity was created in 1905 to solve a problem concerning electromagnetic fields That solution was reached by means of profound changes in fundamental concepts and ideas that considerably affected the whole of physics Moreover when Einstein took gravitation into account he was forced to develop radical changes also in our space time concepts 1916 Relativistic works on heat thermodynamics and elasticity appeared as early as 1911 However general theories having a thermodynamic basis including heat conduction and constitutive equations did not appear in general relativity until about 1955 for fluids and appeared only after 1960 for elastic or more general finitely deformed materials These theories dealt with materials with memory and in this connection some relativistic versions of the principle of material indifference were considered Even more recently relativistic theories incorporating finite deformations for polarizable and magnetizable materials and those in which couple stresses are considered have been formulated A broader description of the development of these relativistic topics is contained in 13 The purpose of this book is to describe the foundations of the general relativistic theories that include constitutive equations and to present some applications mainly to elastic waves of these theories This tract is divided into two parts In the first part only the Eulerian point of view is considered basic equations of general relativity other than constitutive equations are stated in full generality except for couple stresses which are considered in

part 2 Part 1 also thoroughly covers fluids including constitutive equations Lezioni di meccanica razionale: pt. 1, 2
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razionale: 102 pte. Dinamica del sistemi con un numero finito di gradi di liberta Tullio Levi-Civita,1952
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nonholonomic systems technical problems of the directional stability of rolling systems and the general theory of electrical
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